

# ADVANCED FLUID MECHANICS

## Homework 4: Navier Stokes Equation & Boundary Layer

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### 1. Fluid stream impinging a plate

a) The stream potential can also be expressed in Cartesian coordinates as:

$$\psi = U r^2 \sin(2\theta) = U r^2 \cdot 2 \sin\theta \cos\theta = 2U(x^2 + y^2) \frac{xy}{x^2 + y^2} = 2Uxy$$

The velocity in the Cartesian coordinates can be calculated as:

$$v_x = \frac{\partial \psi}{\partial y} = 2Ux \quad \text{and} \quad v_y = -\frac{\partial \psi}{\partial x} = -2Uy$$

At  $y=0$ , we have:

$$v_x = 2Ux \quad \text{and} \quad v_y = 0$$

Hence, it satisfies the boundary condition for an inviscid flow over the plane boundary.

The Stream function can be checked for irrotationality as:

$$\nabla^2 \psi = \frac{\partial^2 \psi}{\partial x^2} + \frac{\partial^2 \psi}{\partial y^2} = 0$$

Hence, the flow is irrotational. The Bernoulli's equation can be applied across the streamlines.

The velocity at the origin is:

$$v_x = 0 \quad \text{and} \quad v_y = 0$$

which is a stagnation point. Assuming the pressure at the origin, the stagnation pressure as the reference pressure  $p_0$ , and in the absence of any body forces, we can apply Bernoulli equation as:

$$\begin{aligned} p_0 &= p_{xy} + \frac{1}{2} \rho v^2 \\ \Rightarrow p_{xy} &= p_0 - \frac{1}{2} \rho v^2 \\ \Rightarrow p_{xy} &= p_0 - \frac{1}{2} \rho \cdot 4U^2(x^2 + y^2) \\ \Rightarrow p_{xy} &= p_0 - 2\rho \cdot U^2 r^2 \end{aligned}$$

b) The Navier Stokes equation for a steady flow can be written as follows:

$$\rho \mathbf{v} \cdot \nabla \mathbf{v} = -\nabla p + \mu \nabla^2 \mathbf{v}$$

Evaluating individual terms:

$$\mathbf{v} \cdot \nabla \mathbf{v} = \left[ v_x \frac{\partial}{\partial x} + v_y \frac{\partial}{\partial y} \right] \mathbf{v} = \begin{bmatrix} 4U^2 x \\ 4U^2 y \end{bmatrix},$$

$$-\nabla p = \begin{bmatrix} -4\rho U^2 x \\ -4\rho U^2 y \end{bmatrix}, \text{ and } \nabla^2 \mathbf{v} = \mathbf{0}$$

The above terms satisfy the Navier-Stokes equation on substitution. Note that the viscous term is zero, and the pressure gradient completely balances the acceleration term.

- c) The modified velocity field with  $u=2Uxf'(y)$  will satisfy the continuity equation under the condition:

$$\begin{aligned} \frac{\partial v_x}{\partial x} + \frac{\partial v_y}{\partial y} &= 0 \\ \Rightarrow 2Uf'(y) + \frac{\partial v_y}{\partial y} &= 0 \\ \Rightarrow v_y &= -2Uf(y) + g(x) \end{aligned}$$

Applying boundary conditions for no-slip condition over the wall, we get:

$$\begin{aligned} v_y(y=0) &= v_x(y=0) = 0 \\ \Rightarrow -2Uf(0) + g(x) &= 0 \quad \& \quad 2Uxf'(0) = 0 \\ \Rightarrow g(x) &= k \quad \& \quad f'(y) = 0, \text{ where } k \text{ is a constant.} \end{aligned}$$

And the constant can be absorbed by the function  $f(y)$ , so  $v_y = -2Uf(y)$

Hence, the boundary conditions are  $f(0)=0$  &  $f'(0)=0$

- d) The y-momentum equation is:

$$\rho \left( v_x \frac{\partial}{\partial x} + v_y \frac{\partial}{\partial y} \right) v_y = -\frac{\partial p}{\partial y} + \mu \left( \frac{\partial^2}{\partial x^2} + \frac{\partial^2}{\partial y^2} \right) v_y$$

On substituting the new velocity field, we obtain:

$$\begin{aligned} \rho \left( 2Uf'(y) \frac{\partial}{\partial x} - 2Uf(y) \frac{\partial}{\partial y} \right) [-2Uf(y)] &= -\frac{\partial p}{\partial y} + \mu \left( \frac{\partial^2}{\partial x^2} + \frac{\partial^2}{\partial y^2} \right) [-2Uf(y)] \\ \Rightarrow 4\rho U^2 f(y) f'(y) &= -\frac{\partial p}{\partial y} - 2\mu U f''(y) \\ \Rightarrow \frac{\partial p}{\partial y} &= -2\mu U f''(y) - 4\rho U^2 f(y) f'(y) \end{aligned}$$

On integrating the above obtained expression for pressure, we have:

$$\begin{aligned} \int \frac{\partial p}{\partial y} dy &= \int [-2\mu U f''(y) - 4\rho U^2 f(y) f'(y)] dy \\ \Rightarrow p_{xy} &= [-2\mu U f'(y) - 2\rho U^2 f^2(y)] + h(x) \end{aligned}$$

The pressure gradient in both velocity potentials must match at large  $y$ . Hence, at large  $y$ , we have:

$$\begin{aligned} \lim_{y \rightarrow \infty} p_{xy} &= p_0 - 2\rho U^2 (x^2 + y^2) \\ \Rightarrow \lim_{y \rightarrow \infty} f^2(y) &= y^2, \text{ and } \lim_{y \rightarrow \infty} [h(x) - 2\mu U f'(y)] = -2\rho U^2 x^2 + p_0 \\ \Rightarrow \lim_{y \rightarrow \infty} f(y) &= y, \text{ and } h(x) - 2\mu U \left[ \lim_{y \rightarrow \infty} f'(y) \right] = -2\rho U^2 x^2 + p_0 \end{aligned}$$

Since,  $f(y)$  approaches  $y$  asymptotically, we must have  $f'(y)$  approach 1 at  $\infty$ .

Hence, the function  $h(x)$  is:

$$h(x) = p_0 + 2\mu U - 2\rho U^2 x^2$$

and the pressure field is:

$$p_{xy} = p_0 - 2\mu U [1 - f'(y)] - 2\rho U^2 [f^2(y) + x^2]$$

e) The x-momentum equation is:

$$\begin{aligned} \rho \left( v_x \frac{\partial}{\partial x} + v_y \frac{\partial}{\partial y} \right) v_x &= -\frac{\partial p}{\partial x} + \mu \left( \frac{\partial^2}{\partial x^2} + \frac{\partial^2}{\partial y^2} \right) v_x \\ \Rightarrow \rho \left[ (2U x f'(y)) \frac{\partial}{\partial x} + (-2U f(y)) \frac{\partial}{\partial y} \right] (2U x f'(y)) \\ &= 4\rho U^2 x + \mu \left( \frac{\partial^2}{\partial x^2} + \frac{\partial^2}{\partial y^2} \right) (2U x f'(y)) \\ \Rightarrow 4\rho U^2 x [f'^2(y) - f(y) f''(y)] &= 4\rho U^2 x + 2\mu U x f'''(y) \\ \Rightarrow f''' + \frac{2\rho U}{\mu} (f \cdot f'' - f'^2 + 1) &= 0 \end{aligned}$$

Since, this a 3<sup>rd</sup> degree ODE, and we have three boundary conditions, the system has a unique solution.

## 2. Kármán-Pohlhausen approximation to Boundary Layer

The given velocity profile is:

$$\frac{u}{U} = a + b \left( \frac{y}{\delta} \right) + c \left( \frac{y}{\delta} \right)^2$$

and the boundary conditions :

$$\begin{aligned} u &= 0 \text{ at } y=0 \\ u &= U, \quad \frac{\partial u}{\partial y} = 0 \text{ at } y=\delta \end{aligned}$$

On substituting the given velocity profile into the given boundary conditions, we get:

$$\frac{u}{U} = 2 \frac{y}{\delta} - \left( \frac{y}{\delta} \right)^2$$

Evaluate momentum thickness of the boundary layer using:

$$\begin{aligned} \frac{\theta}{\delta} &= \int_0^1 \frac{u}{U} \left( 1 - \frac{u}{U} \right) d \left( \frac{y}{\delta} \right) \\ \Rightarrow \theta &= \frac{2}{15} \delta \end{aligned}$$

Also, the shear stress at  $y=0$  can be evaluated as:

$$\begin{aligned} \frac{\tau_0}{\rho} &= \frac{2\nu U}{\delta} \\ \frac{d}{dx} U^2 \theta &= \frac{\tau_0}{\rho} \end{aligned}$$

since  $U$  does not vary with  $x$ ,

$$\Rightarrow U^2 \frac{d}{dx} \left( \frac{2}{15} \delta \right) = \frac{2\nu U}{\delta}$$

$$\Rightarrow \delta^2 = 30 \frac{\nu x}{U}$$

$$\Rightarrow \frac{\delta}{x} = \frac{5.4772}{\sqrt{\text{Re}_x}}$$

We have the solution for boundary layer thickness and hence, the velocity profile in the boundary layer.

On comparison with Blassius solution and General momentum solution with a 3<sup>rd</sup> degree polynomic velocity profile, we can see that the  $\delta/x$  is still proportional to  $\text{Re}^{-1/2}$ . But the constant of relation changes. For Blassius solution and with cubic polynomial, the boundary layer thickness is:

$$\frac{\delta}{x} = \frac{5}{\sqrt{\text{Re}_x}} \quad (\text{Blassius solution})$$

$$\frac{\delta}{x} = \frac{4.64}{\sqrt{\text{Re}_x}} \quad (\text{cubic profile})$$

Hence,  $\delta_{cubic} < \delta_{Blassius} < \delta_{quadratic}$

The shear stress on the plate for all three cases are ass follows:

$$\frac{\tau_0}{\rho} = \frac{0.7303}{\sqrt{\text{Re}_x}} \quad (\text{quadratic profile})$$

$$\frac{\tau_0}{\rho} = \frac{0.6466}{\sqrt{\text{Re}_x}} \quad (\text{cubic profile})$$

$$\frac{\tau_0}{1/2\rho U^2} = \frac{0.664}{\sqrt{\text{Re}_x}} \quad (\text{Blassius solution}).$$

$$\Rightarrow \tau_{cubic} < \tau_{Blassius} < \tau_{quadratic}$$

Hence, it can concluded that the solution obtained using the quadratic boundary layer is thicker and creates more drag.

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