

Assignment 2.1

On “FEM Modelling: Introduction”:

1. Identify the symmetry and antisymmetry lines in the two-dimensional problems illustrated in the figure. They are:
 - (a) a circular disk under two diametrically opposite point forces (the famous “Brazilian test” for concrete)
 - (b) the same disk under two diametrically opposite force pairs
 - (c) a clamped semiannulus under a force pair oriented as shown
 - (d) a stretched rectangular plate with a central circular hole.
 - (e) and (f) are half-planes under concentrated loads.

2. Having identified those symmetry/antisymmetry lines, state whether it is possible to cut the complete structure to one half or one quarter before laying out a finite element mesh. Then draw a coarse FE mesh indicating, with rollers or fixed supports, which kind of displacement BCs you would specify on the symmetry or antisymmetry lines.

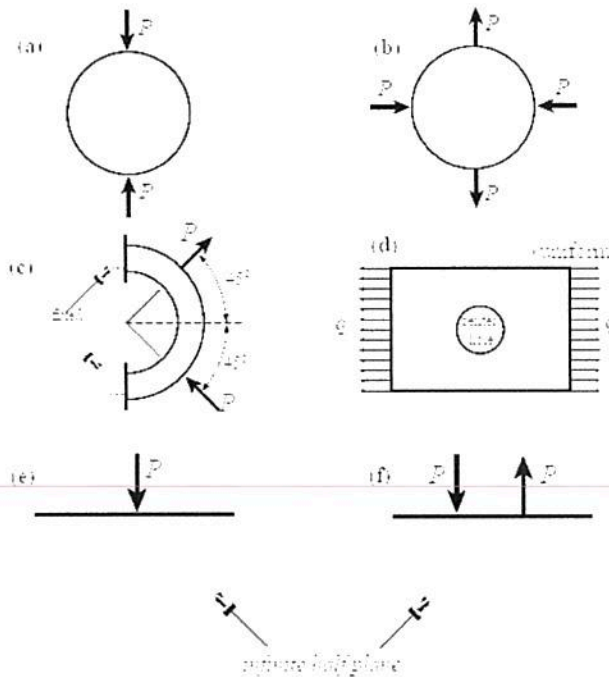


Figure 2.1.- Problems for assignment 2.1

Assignment 2.2

Explain the difference between “*Verification*” and “*Validation*” in the context of the FEM-Modelling procedure.

Assignment 2.3

On “Variational Formulation”:

1. A tapered bar element of length l and areas A_i and A_j with A interpolated as

$$A = A_i(1 - \xi) + A_j\xi$$

and constant density ρ rotates on a plane at uniform angular velocity ω (rad/sec) about node i . Taking axis x along the rotating bar with origin at node i , the centrifugal axial force is $q(x) = \rho A \omega^2 x$ along the length in which x is the longitudinal coordinate $x = x^e$.

Find the consistent node forces as functions of ρ , A_i , A_j , ω and l , and specialize the result to the prismatic bar $A = A_i = A_j$.

Date of Assignment: 17 / 02 / 2020

Date of Submission: 24 / 02 / 2020

The assignment must be submitted as a pdf file named **As2-Surname.pdf** to the CIMNE virtual center.

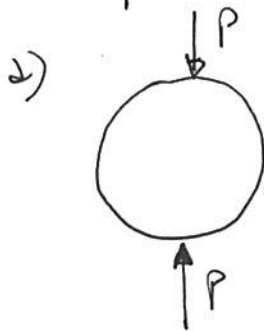
$$\begin{aligned} F &= m \cdot a \\ &= m \cdot \frac{v^2}{r} \end{aligned}$$

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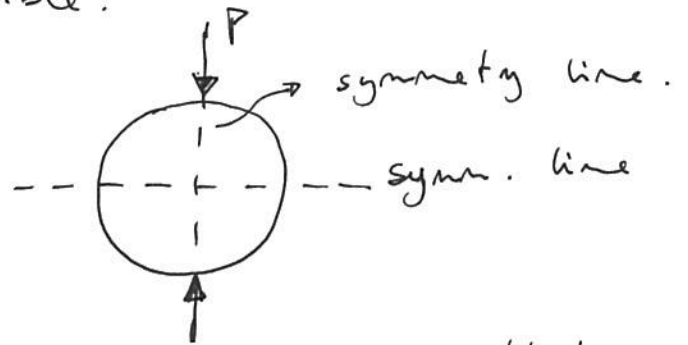
Assignment 2

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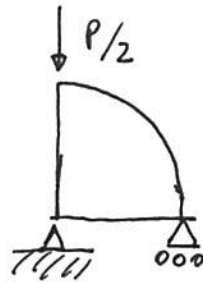
2.1 Identify symmetry lines and use them to cut the structure to a half or a quarter as possible.



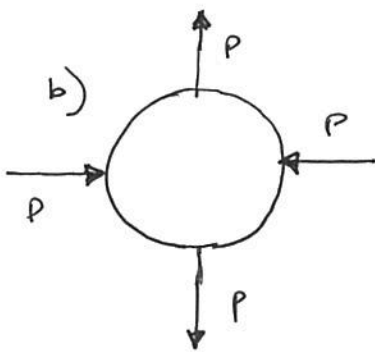
answer:



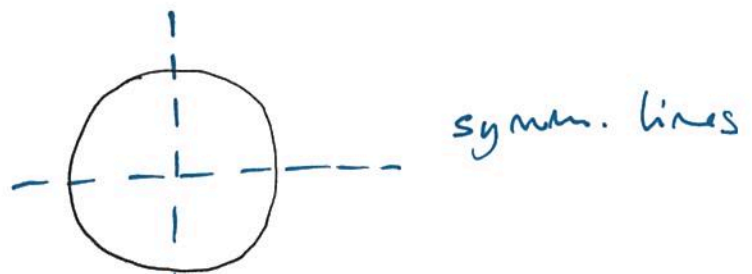
it can be appropriately modelled by $1/4$ using half the load:



the center will not move (forces cancel out)

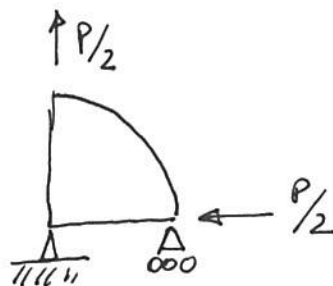


answer

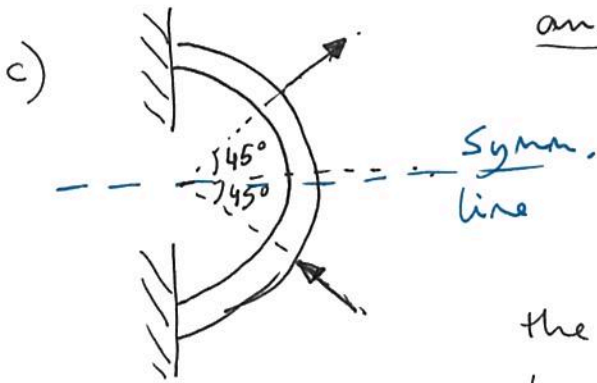


as in the prior case, stresses cancel out in the center (center is fixed)

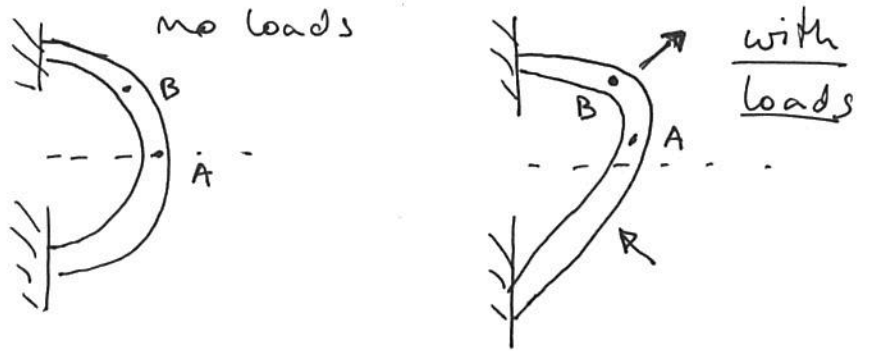
the



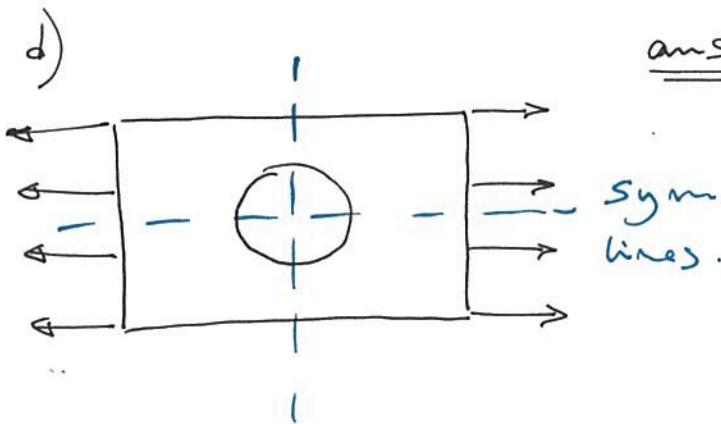
(2)



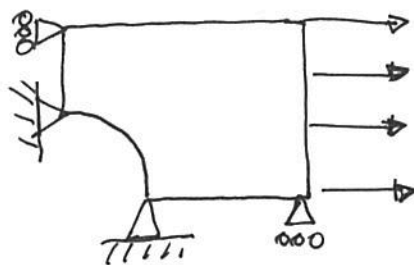
answer: although there is a symmetry line the forces do not cancel out. furthermore the deformation will not be symmetric:

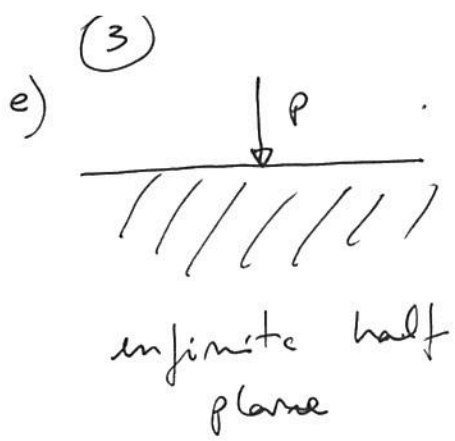


therefore, the system needs to be modelled in full

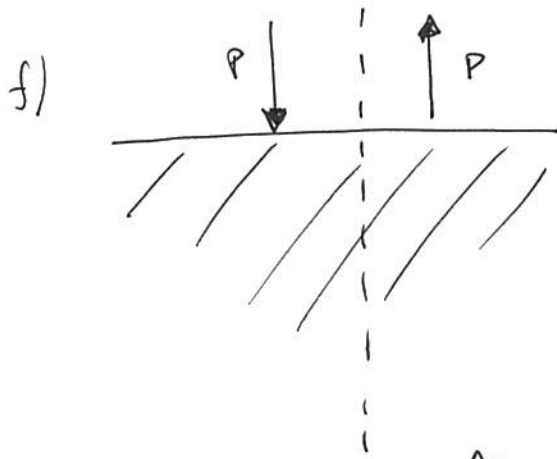
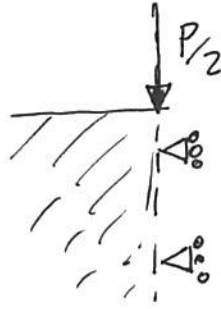


answer: the system has two axes of symmetry, along which displacements cancel. therefore $\frac{1}{2}$ of the system is enough to capture all deformations

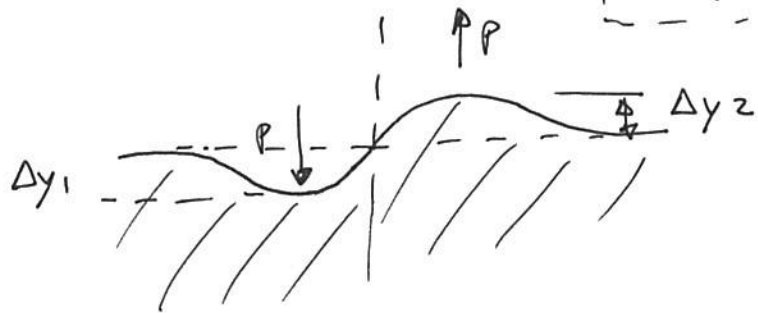




answer: there is a vertical line of symmetry. deformations and stresses will be the same on both side. model $\frac{1}{2}$ of the system.



answer there is an anti symmetry axis. after deformation it would look like the



I'm not sure at all that $\Delta y_1 = \Delta y_2$ (that is that deformations are anti-symmetric. I would advise modelling the whole system.

4

2.2

Verification: is the process of ensuring that the discrete solution arrived at is a 'good enough' answer to the mathematical problem initially proposed:

- Does the discretization represent the problem correctly (convergence, consistency)?
- Did our method correctly solve the discrete problem?

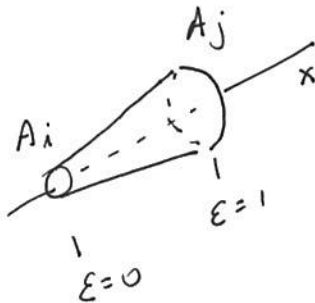
Validation: compare the model predictions with the experimental results. the sources of error could also be in: unknown properties of the materials or inadequate physical modelling. verification would not capture those two sources of error! Hence we need validation.

5

2.3

Find model forces on a rotating bar of variable section.

$$A(x) = A_i (1 - \epsilon) + A_j (\epsilon)$$



take a cross sectional slice of the bar



neglecting gravity, the body force at the center of each element is

$$f^e = m^e \ddot{a} = m^e \frac{v^2}{r} = A^e \rho \frac{v^2}{r} = A^e \rho \omega^2 r^e$$

where $v = \omega r^e$ is the tangential velocity. and r^e is the distance of the center of the element to the axis of rotation. The coordinate ϵ affect the radius. Assuming equilateral triangles of side a



$$A^e = \frac{\sqrt{3}}{4} a^2 \Rightarrow a = 1.516 A^e$$

$$r = h = r^e = \frac{a}{2 \cos 30} = 3.24 a$$

$$= 4.9 A^e$$

(b) Applying the forces on each mode
equally

$$f_i^e = \frac{A^e \rho \omega^2 r_e}{3} = [A_i(1-\epsilon) + A_j(\epsilon)]^2 \rho \omega^2. \quad (1.6)$$

mode

with $r_e = 4.9 A^e = 4.9(A_i(1-\epsilon) + A_j\epsilon)$

this way, the force is equally distributed
in the three modes.